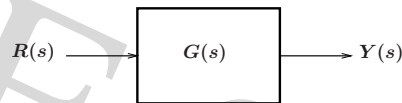


- Pole position and time domain relationships
- Typical time domain specifications
- First-order systems
- Second-order systems

- Typical time domain specifications
 - overshoot
 - settling time
 - rise time
 - time to peak
- Typical frequency domain specifications
 - natural frequency
 - damped frequency
 - damping ratio
 - bandwidth

Poles Affect Time Behavior

- Plant : $G(s)$



$$G(s) = k \frac{(s - z_1) \dots (s - z_m)}{(s - p_1) \dots (s - p_n)}$$

- Assuming no duplicated poles and $m < n$,

$$G(s) = \frac{R_1}{s - p_1} + \dots + \frac{R_n}{s - p_n}$$

Poles Affect Time Behavior

- Impulse response.

$$\mathcal{L}^{-1} \Rightarrow g(t) = \left(R_1 e^{p_1 t} + \dots + R_n e^{p_n t} \right) u(t)$$

- Q : If systems are high-order, do we need to compute all the terms to get the response?

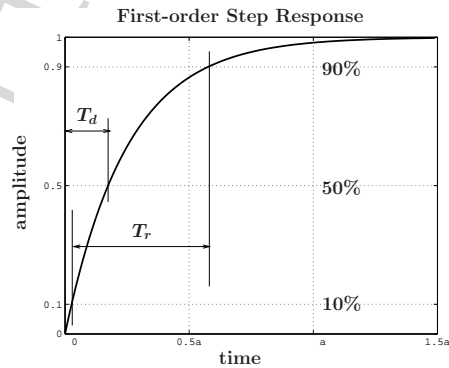
A : Not always. High-order systems sometimes behave like low-order systems.

- Study low-order systems first.

First-order Systems

- Consider

$$G(s) = \frac{a}{s + a}$$



First-order Systems

- Q : What is the DC gain?
- Q : What is the steady-state output for $x(t) = u(t)$?

- Definitions.

rise time $\triangleq T_r$ time for the step response to go from 10% to 90%.

delay time $\triangleq T_d$ time for the step response to reach 50% of the final value.

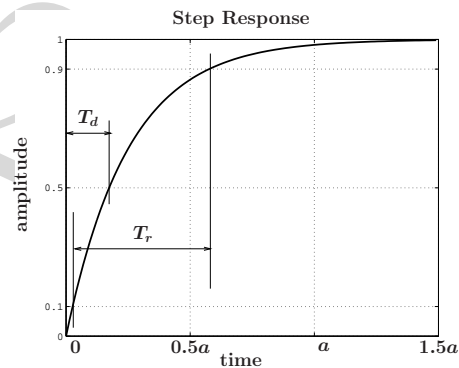
First-order Systems

- Relationships of T_r and T_d with system parameter a .

$$T_r = \frac{2.2}{a} = 2.2\tau \quad T_d = \frac{0.69}{a} = 0.69\tau$$

- Octave command.

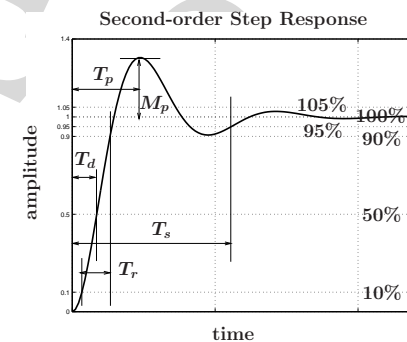
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>> step(2, [1 2])
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Second-order Systems

- Consider

$$G(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$



• What is the DC gain?

The DC gain is the steady-state output of the system in response to a step input.

• Definitions.

$M_p \triangleq$ peak overshoot / max overshoot

$T_p \triangleq$ time to peak overshoot

$T_s \triangleq$ settling time

• System characteristics.

ζ	Description	Poles
$\zeta > 1$	real poles overdamped	LHP
$\zeta = 1$	real identical poles critically damped	LHP
$0 < \zeta < 1$	complex conjugate poles underdamped	LHP
$\zeta = 0$	imaginary poles undamped	imaginary axis
$\zeta < 0$	poles with positive real part negatively damped	RHP

• Poles of the second-order system.

$$s = -\zeta\omega_n \pm j\omega_n\sqrt{1 - \zeta^2}$$

• System parameters.

ω_n : natural frequency

ζ : damping factor

• Octave command.

>> step(2, [1 1 2])

• Step response.

$$Y(s) = G(s) \frac{1}{s} = \frac{1}{s} - \frac{s + 2\zeta\omega_n}{(s + \zeta\omega_n)^2 + \omega_n^2(1 - \zeta^2)}$$

$$\Rightarrow y(t) = 1 - \frac{e^{-\frac{t}{\tau}}}{\sqrt{1 - \zeta^2}} \cos(\omega_d t - \rho_d), t \geq 0$$

where

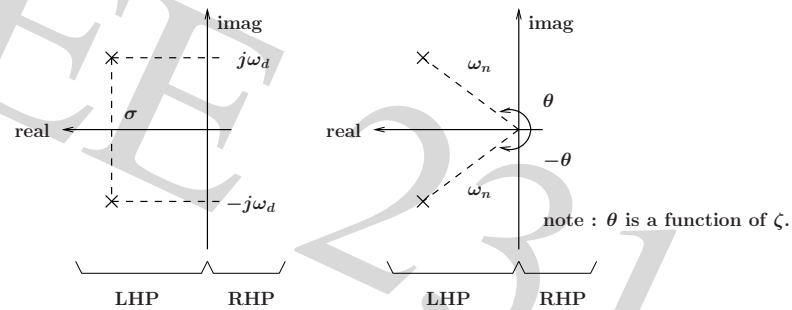
$$\sigma = -\zeta\omega_n$$

$$\tau = \frac{1}{|\sigma|}$$

$$\rho_d = \sin^{-1} \zeta$$

$$\omega_d = \omega_n \sqrt{1 - \zeta^2} \leftarrow \text{damped } \omega$$

- Location of poles in the s-plane.



- System parameters are the starting point in the design.
- The behavior of first-order systems is dictated by one parameter.
- Second-order systems behavior is governed by two parameters.
- Two ways of looking at the poles in the s-plane.